

“ ELIMINATION OF REPETITIVE THRUST BEARING FAILURES ON A PROCESS AIR COMPRESSOR”

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INDIA**

**35TH TURBOMACHINERY SYMPOSIUM' 2006
THE TEXAS A & M UNIVERSITY SYSTEM**

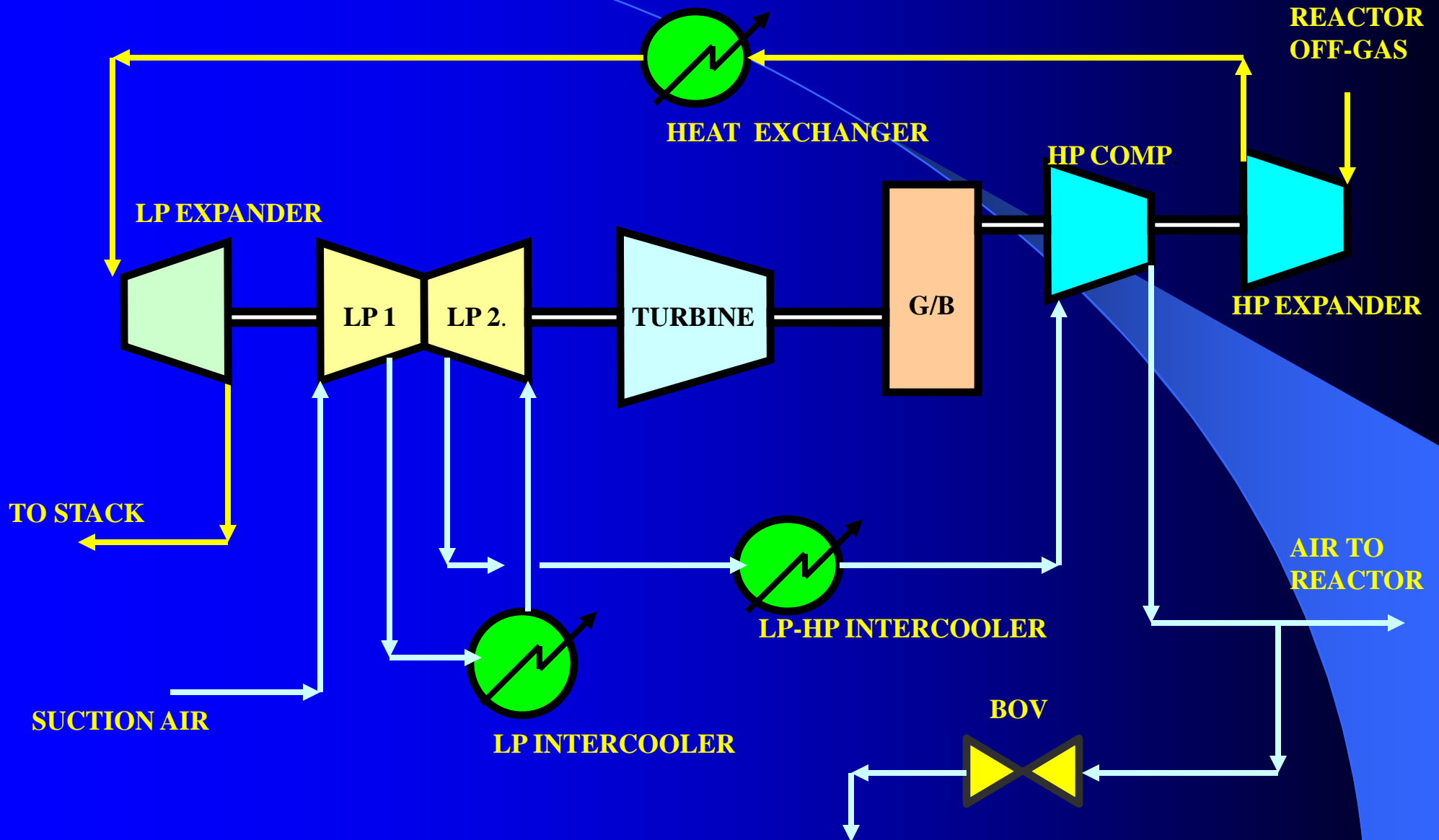
BRIEF DESCRIPTION OF PROBLEM

- ❖ Repetitive failure of thrust bearing in a Turbo Driven Process Air Centrifugal Compressor.
- ❖ 3 failures in 2004-2005.
- ❖ The downtime for replacement of this bearing was amounting to a loss of **US \$ 88,888 (Rs. 4 million) / failure.**

EQUIPMENT UNDER STUDY

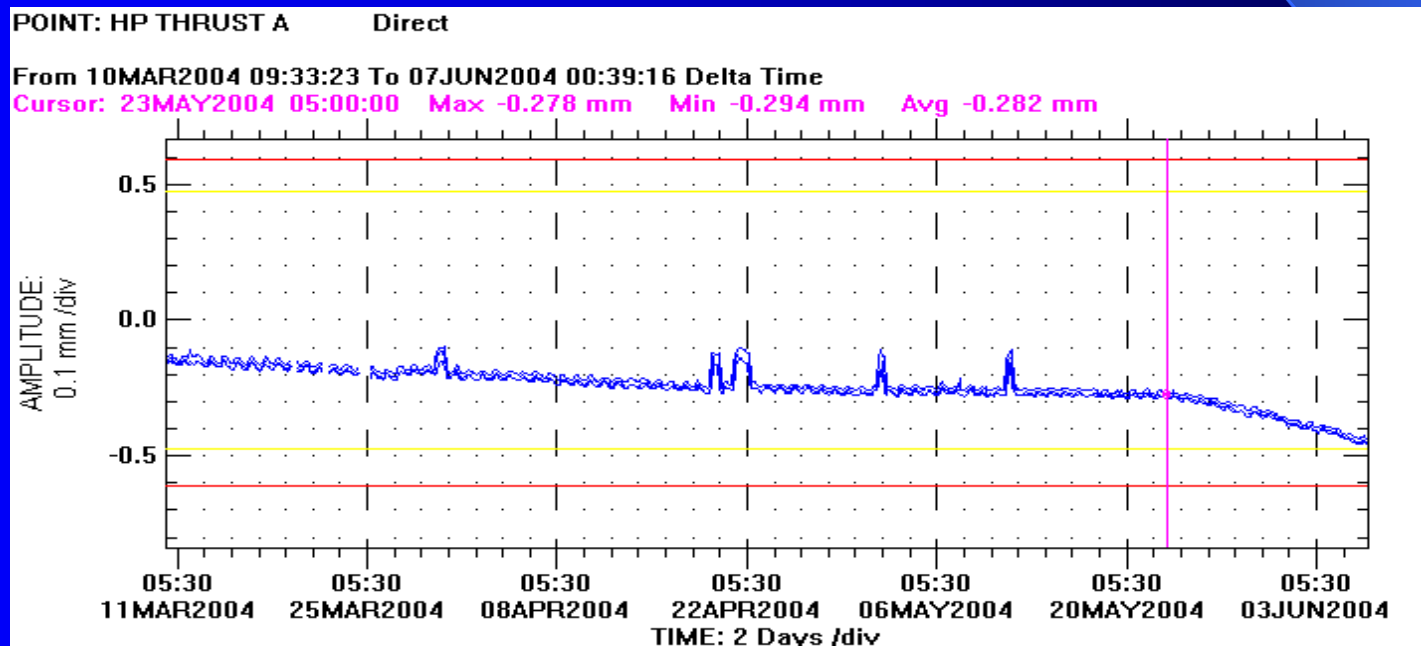
- **NAME** : PROCESS AIR COMPRESSOR.
- **SUC. FLOW RATE** : 63340 M³/HR (37280 CFM)
- **DISCH. PRESSURE** : 25.2 BAR A
- **DISCHARGE TEMP** : 194⁰C (381⁰F)
- **SPEED** : 13,920 RPM
- **1ST / 2ND CRITICAL** : 8276 / 17576 RPM
- **DESIGN PR. / TEMP.** : 27.5 BAR A / 250 ⁰C (482 ⁰F)
- **DRIVE** : TURBINE (CONDENSING TYPE- 16 HP & 6 LP STAGES, RATED / MAX. OUTPUT- 6.6 MW / 8.9MW)

MACHINE TRAIN DIAGRAM



HISTORY

- ❖ HP Compressor active side thrust bearing was replaced during the annual shutdown (Jan'2004), as severe pitting was found on its pads .
- ❖ 5 months thereafter, increasing trend with a sharp spurt, observed in the axial displacement of the HP compressor.

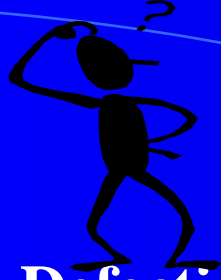


PARAMETER OBSERVATIONS

- No appreciable change in the process parameters such as flow or pressure.
- Increase of @ $5-6^{\circ}\text{C}$ ($9^{\circ}\text{F} - 10^{\circ}\text{F}$) in the active side thrust bearing temperature.
- Vibrations at the journal bearings normal, with no abnormality in the spectrums & orbit plots.

WHAT COULD BE PROBABLE CAUSE??





PROBABLE CAUSES OF FAILURE

❖ Defective bearings ?

- **Ruled out as bearings were OEM supplied.**

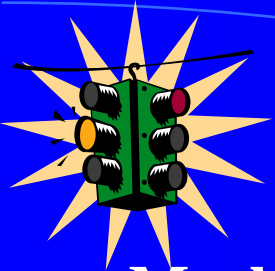
❖ Improper lube oil supply ?

- **Ruled out as journal bearing temp. Were normal.**

❖ Faulty shaft grounding ?

❖ Visual inspection of the dual carbon shaft grounding brush not possible due to inaccessibility.

❖ No current observed in the Earthing terminal wire, when checked with milli- Amp Tong tester.



OBSERVATIONS

Machine was shut down for inspection/ replacement of thrust bearing & grounding brushes . Following observations were made-

- ❖ Shaft grounding brushes worn out & jammed in the carrier.



OBERVATIONS

- ❖ Uniform wear out & “**Frosting**” marks along with oil varnishing observed on Active Thrust bearing pads .





OBSERVATIONS

- ❖ Wear out of **0.35mm (13.77 mils)** observed in the active side pads.
- ❖ **No Abnormality or Wear** in the inactive side bearing .


IMMEDIATE ACTIONS TAKEN

- ❖ Active side Bearing replaced with new one.
- ❖ Brush carrier thoroughly cleaned & brushes made free.
- ❖ Axial float adjusted to **0.2 mm (7.87 mils)**.
- ❖ Machine boxed up and taken in line. **All parameters found normal.**
- ❖ Axial displacement recorded to be **- 0.18 mm (-7.08 mils)** { as against **- 0.46mm (-18.11 mils)** at the time of **Shutdown** }
- ❖ **1mA** current was measured at earthing terminal.

PROBLEM WITH CARBON BRUSHES

- Carbon Brushes By Design Have Following Disadvantages-
 1. Intended for continuous, higher current densities (**60-80 amperes per sq.Inch**).
 2. At low current densities (**less than 40 amperes per sq.Inch**), they develop non-conducting glaze on the contact surfaces, impairing the ability of the brush to properly function.
 3. Brushes "gum-up" in presence of oil and / or dust found in the field and hence require frequent cleaning.
 4. Besides the above disadvantages, in this case, even the inspection of the brushes called for **an Expensive Machine Downtime**.

MECHANISM OF FAILURE

- ❖ In condensing turbines the last stages are subjected to saturated steam.
- ❖ The brushing effect of the water droplets (condensate) across the blades results in development of an electrostatic charge that builds up on the rotor.
- ❖ Charge periodically discharges to the ground through the path of least resistance.
- ❖ Dissipation of rotor voltage across an oil film bearing produces microscopic pits, resulting in loss of babbit, giving it a “**Frosted**”  appearance. Local overheating due to spark results in **Varnishing** of the oil.

MECHANISM OF FAILURE

- ❖ Voltages generated in the turbine rotor are transmitted throughout machine train. Failure can occur on the bearing away from the source of generation.
- ❖ Residual magnetism in the rotor can also be a source of generation of stray electric currents. **Since the slow roll readings and vibrations at operating speed were normal , the chances of residual magnetism in the rotor was ruled out.**

Thus this failure was attributed to the following reasons –

- **Design :- Faulty shaft grounding brushes**
- **Supervision :- No regular check to ensure functioning of brushes.**



RECOMMENDATIONS

- ❖ **Replace brushes with improved design,**
 - ❖ **To enable inspection &**
 - ❖ **Replacement without machine stoppage.**
- ❖ **Regular monitoring of shaft grounding current to ensure proper functioning of the brushes.**



RELIABLE SOLUTION

- ❖ **Bristle Type Brushes By Design Have The Following Advantages-**
- 1. Gold plated, Silver alloy bristle head brushes offer better contact and Very low contact resistance (6 milli-ohm).**
 - 2. Bristle construction ensures sufficient contact and proper functioning even in presence of oil and/or dust.**
 - 3. Mounting system has a wear indicator.**
 - 4. Replacement of brushes with no stoppage of equipment.**
 - 5. Monitoring system with alarm to indicate malfunctioning of any brush.**

“ The Benefits of Reliability to the P-F Curve”

Point where Increasing axial displacement of the Compressor rotor was noticed by Online Vibration Monitoring System

Inspection of Carbon Grounding brushes should have been planned & scheduled

Bearings damaged due to “Frosting” due to Electrostatic discharge

Bearing should have been scheduled and replaced

Other Components identified as failed

PRO-ACTIVE

Cost to Upgrade Grounding Brushes- **US \$ 44,444 OR (Rs.2 Million)**

PREDICTIVE

Cost to Repair & Replace components

US \$ 88,888 OR (Rs.4 Million)

REACTIVE

Restoration Cost -

US \$ 4,44,444 OR (Rs.20 Million)

Compressor Trips causing production loss

INCREASED RELIABILITY

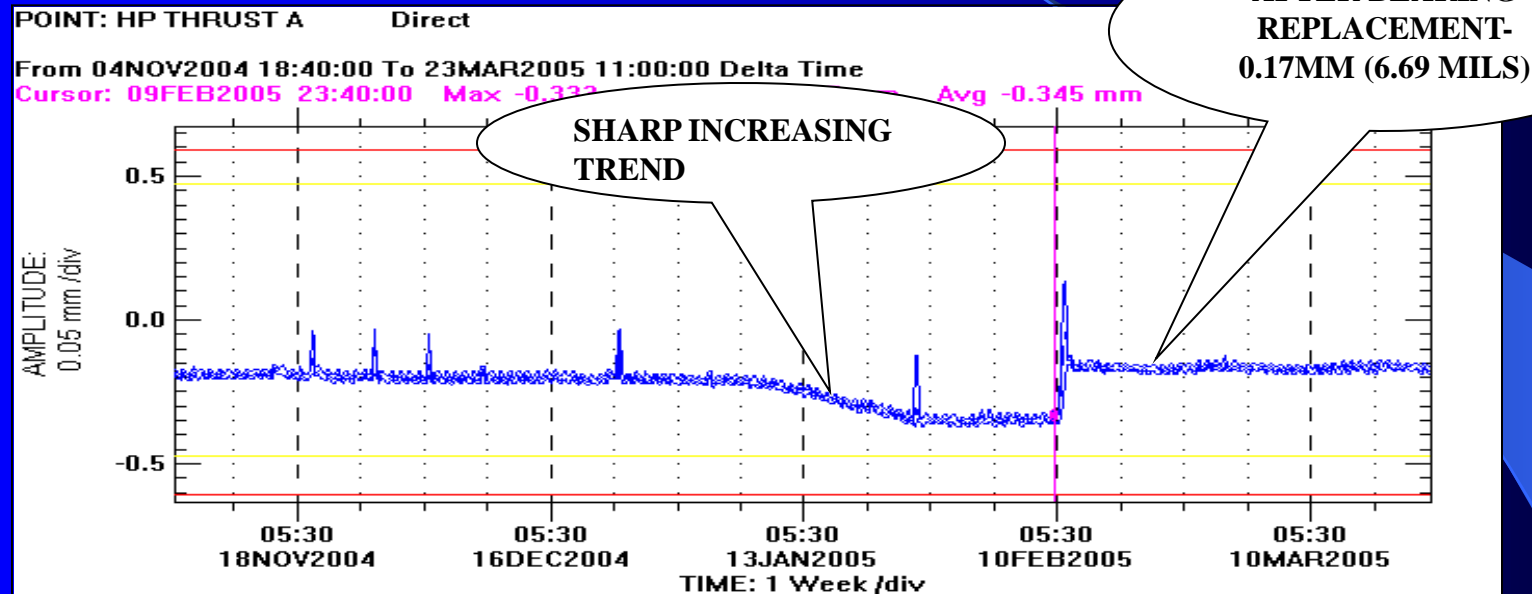
DECREASED RELIABILITY

DECREASED COST

INCREASED COST

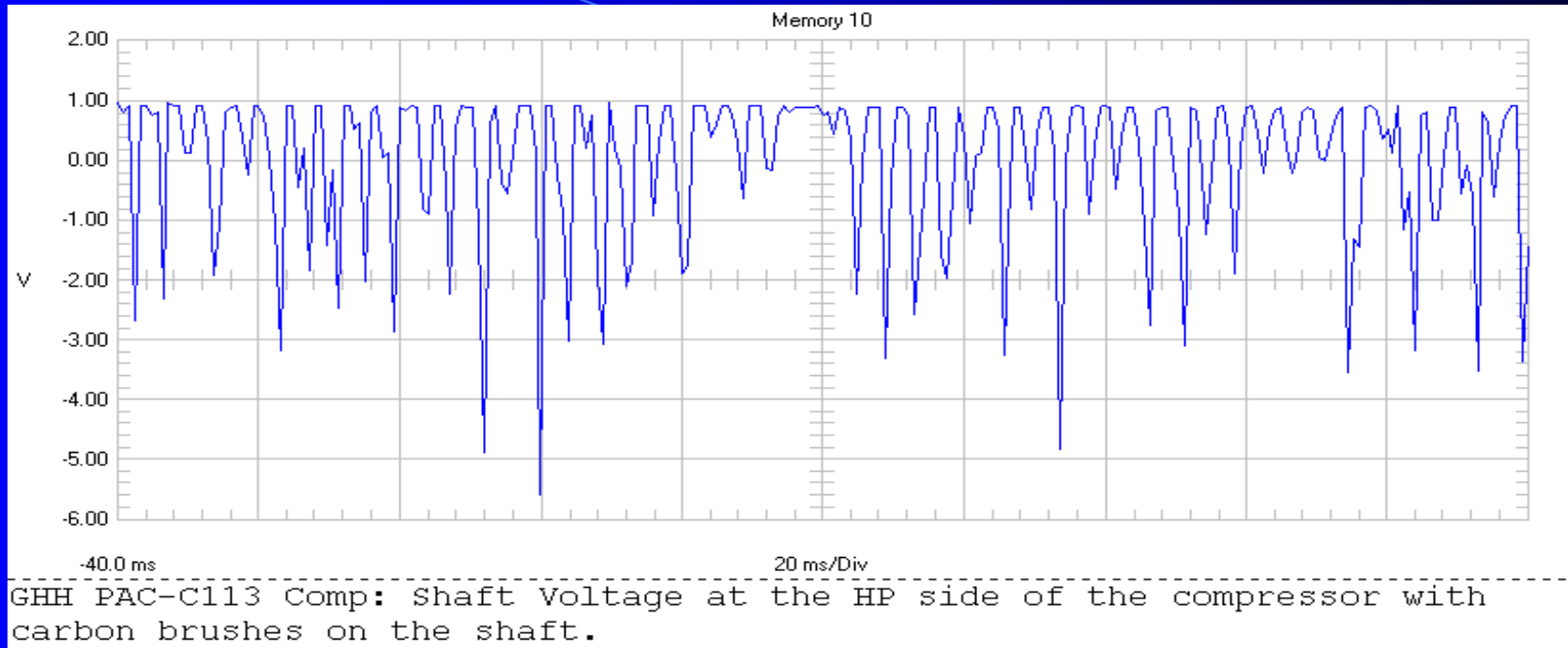
REPETITIVE FAILURE

- Similar increasing trend with a sharp spurt, observed in the axial displacement.



- Observations similar to the earlier failure were made during the planned shutdown .
 - Carbon brushes “gummed-up” or jammed.
 - “Frosting” with pad thickness reduction.

SHAFT VOLTAGE TRACES WITH CARBON BRUSHES

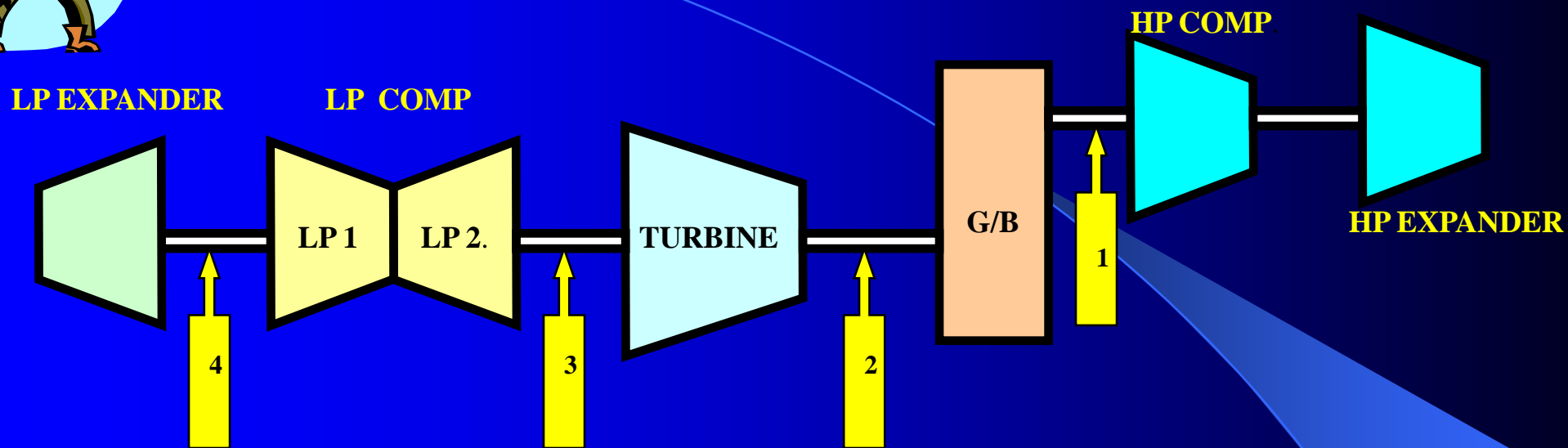


- ❖ Traces taken with Flukescope meter & portable gold plated brush, adjacent to the respective carbon brushes.
- ❖ Shaft voltage of 6V pk-pk was measured between the HP compressor and gearbox

Carbon brushes were ineffective in reducing potential between shaft and bearings, on that specific shaft, to below 50mV.



INSTALLATION OF BRUSHES

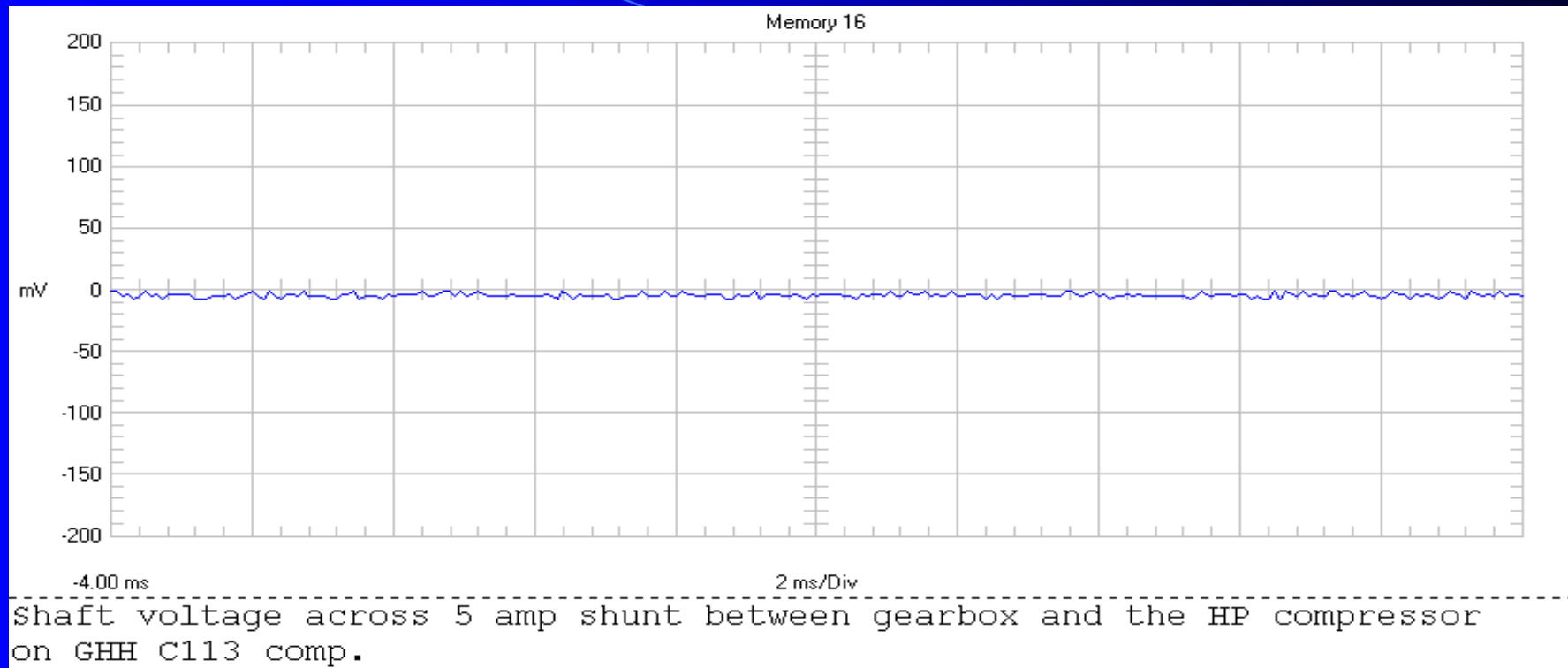


SILVER + GOLD COMPOSITE BRISTLE TYPE BRUSH

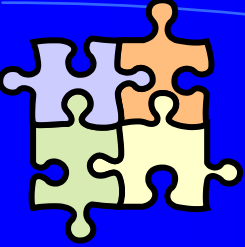


REPLACEABLE
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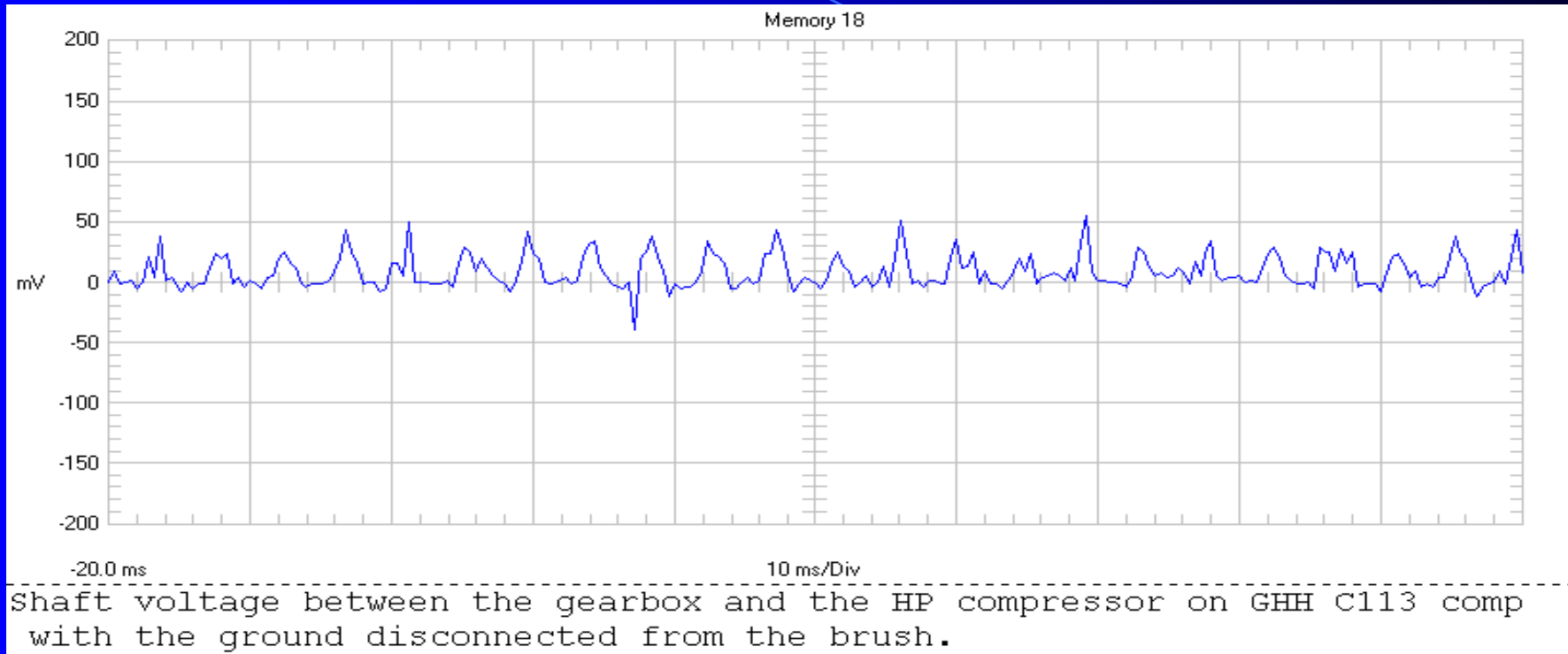
SHAFT VOLTAGE TRACE WITH NEW BRUSHES



- ❖ Voltage traces taken across 5A shunts at different brush locations with the brushes connected to the ground to determine the shaft current.
- ❖ Shaft voltages dropped to below 50mV with a current flow of approx. 0.2A

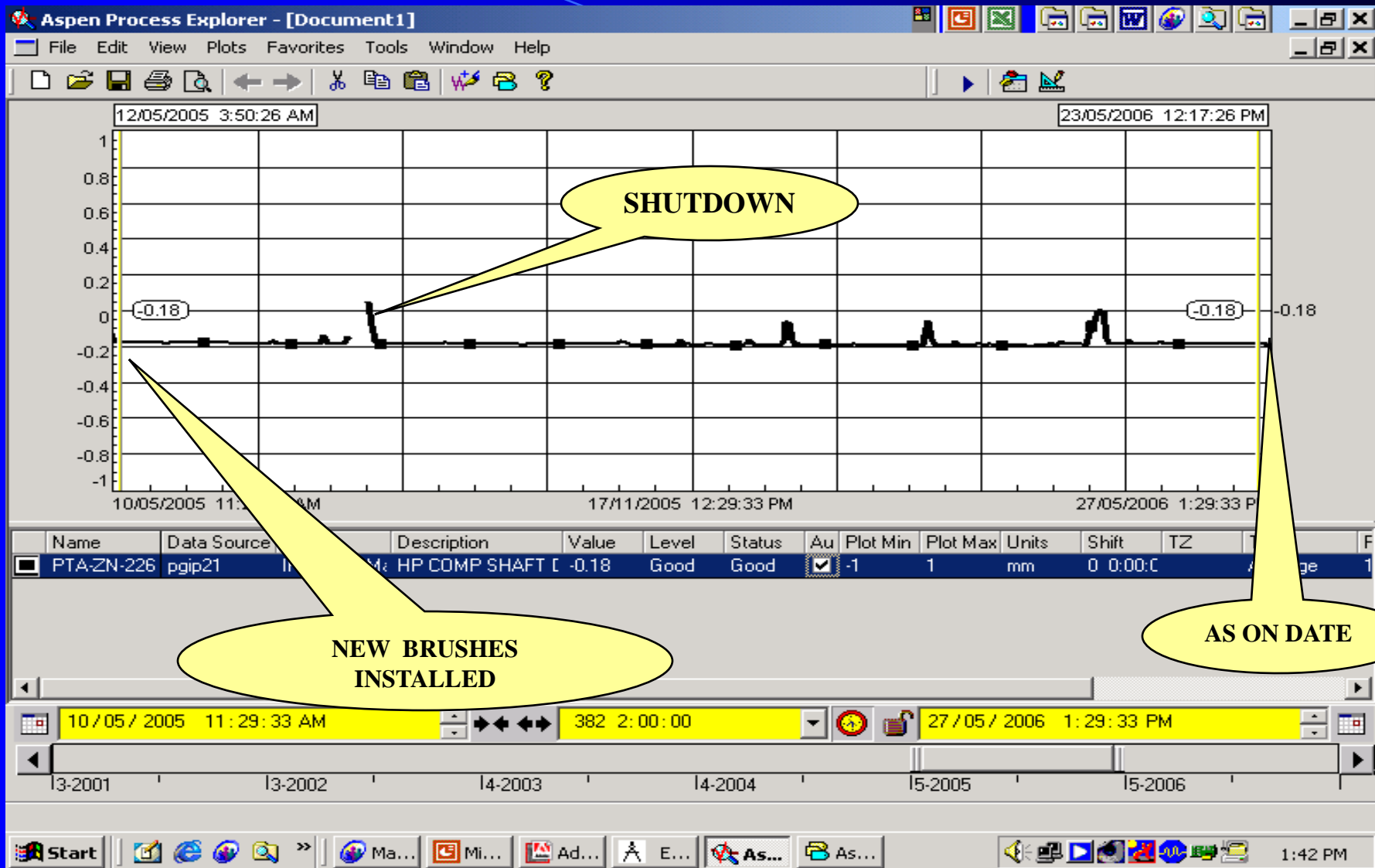


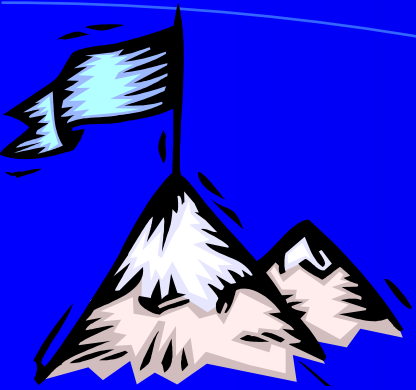
SHAFT VOLTAGE TRACE WITH ONE BRUSH DISCONNECTED



❖ Shaft voltage spikes below 50mV were generated, probably due to some residual magnetism.

AXIAL SHAFT DISPLACEMENT TREND





LEARNINGS

1. **Electrostatic shaft voltages are possible on machine trains driven by condensing steam turbines. These voltages are controllable by isolation and/or grounding of the rotating assemblies.**
2. **Electromagnetic shaft voltages appear when rotating magnetic fields are generated by magnetized machine elements and can only be eliminated by degaussing the magnetized parts.**



THANK YOU